

VOLKSWAGEN

AKTIENGESELLSCHAFT

KONZERNFORSCHUNG



# Aeroacoustic optimisation by means of adjoint sensitivity maps

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## AGENDA

- Continuous adjoint method for external aerodynamics
- Physical mechanisms of sound propagation to the interior
- OpenFOAM-based process for interior noise prediction
- Computing the adjoint aeroacoustic sensitivity maps
- Conclusions & future steps

## ACKNOWLEDGEMENTS

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# THE CONTINUOUS ADJOINT METHOD

## FORMULATION FOR NS-EQUATIONS

- Optimization problem:

minimize the objective function:  $J = J(\vec{v}, p, b)$

subject to:  $\vec{R}(\vec{v}, p, b) = \vec{0}$

- Augmented objective function:  $L = J + \int_{\Omega} q R^p d\Omega + \int_{\Omega} \vec{u} \cdot \vec{R}^v d\Omega$

$$\delta_{\vec{v}, p} = 0$$

- Variation:  $\delta L = \delta_{\vec{v}, p} \left( J + \int_{\Omega} q R^p d\Omega + \int_{\Omega} \vec{u} \cdot \vec{R}^v d\Omega \right) + \delta_b \left( J + \int_{\Omega} q R^p d\Omega + \int_{\Omega} \vec{u} \cdot \vec{R}^v d\Omega \right)$

**Flow (primal) Equations**

$$\begin{cases} \vec{R}(\vec{v}, p, b) = \\ R^p = \nabla \cdot \vec{v} = 0 \\ \vec{R}^v = \frac{\partial \vec{v}}{\partial t} + (\vec{v} \cdot \nabla) \vec{v} - \nabla \cdot \boldsymbol{\tau} + \nabla p = \vec{0} \end{cases}$$

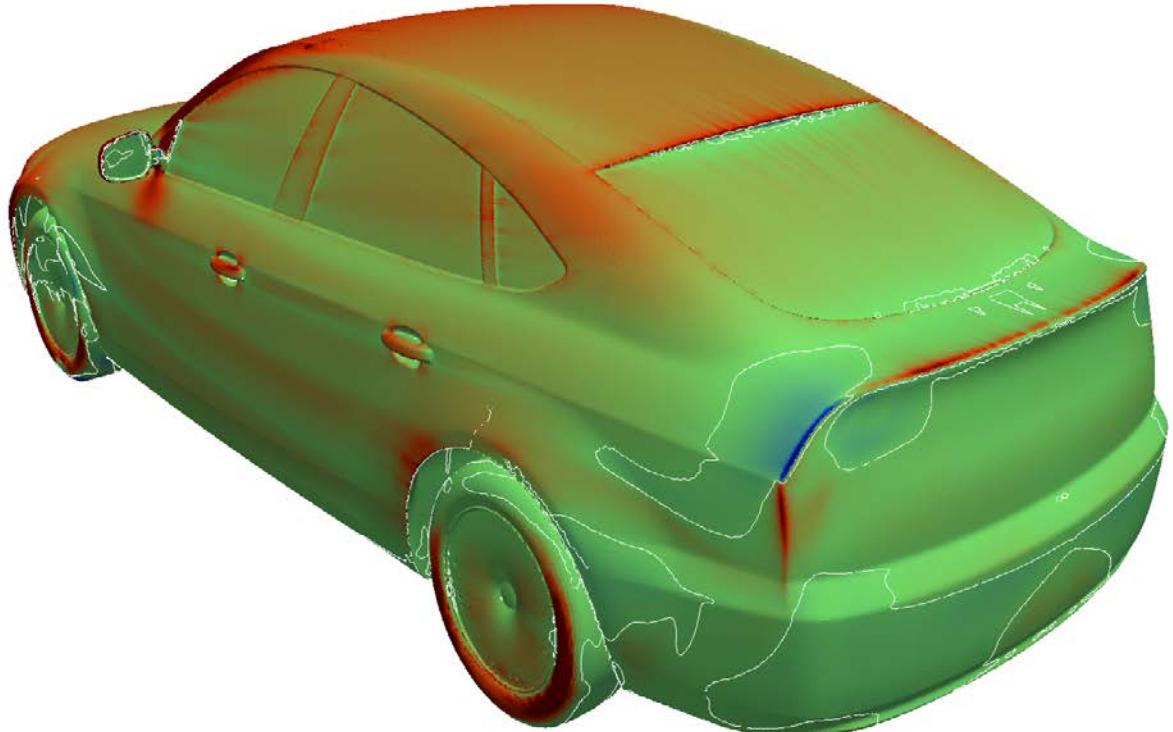
### Adjoint Equations

$$\vec{R}(\vec{u}, q) = \begin{cases} R^q = \nabla \cdot \vec{u} = 0 \\ \vec{R}^u = -\frac{\partial \vec{u}}{\partial t} + -(\vec{v} \cdot \nabla) \vec{u} - \nabla \vec{u} \cdot \vec{v} - \nabla \cdot \boldsymbol{\tau}^a + \nabla q = \vec{0} \end{cases}$$

### Sensitivity Derivatives:

$$\frac{\delta L}{\delta b} = \int_S -\nu \frac{\partial \vec{u}}{\partial n} \cdot \frac{\partial \vec{v}}{\partial n} + [...] dS$$

# THE CONTINUOUS ADJOINT METHOD



## Flow (primal) Equations

$$\begin{cases} \vec{R}(\vec{v}, p, b) = \\ R^p = \nabla \cdot \vec{v} = 0 \\ \vec{R}^v = \frac{\partial \vec{v}}{\partial t} + (\vec{v} \cdot \nabla) \vec{v} - \nabla \cdot \boldsymbol{\tau} + \nabla p = \vec{0} \end{cases}$$

$$\int_{\Omega} \vec{u} \cdot \vec{R}^v d\Omega + \delta_b \left( J + \int_{\Omega} q R^p d\Omega + \int_{\Omega} \vec{u} \cdot \vec{R}^v d\Omega \right)$$

← →

## Adjoint Equations

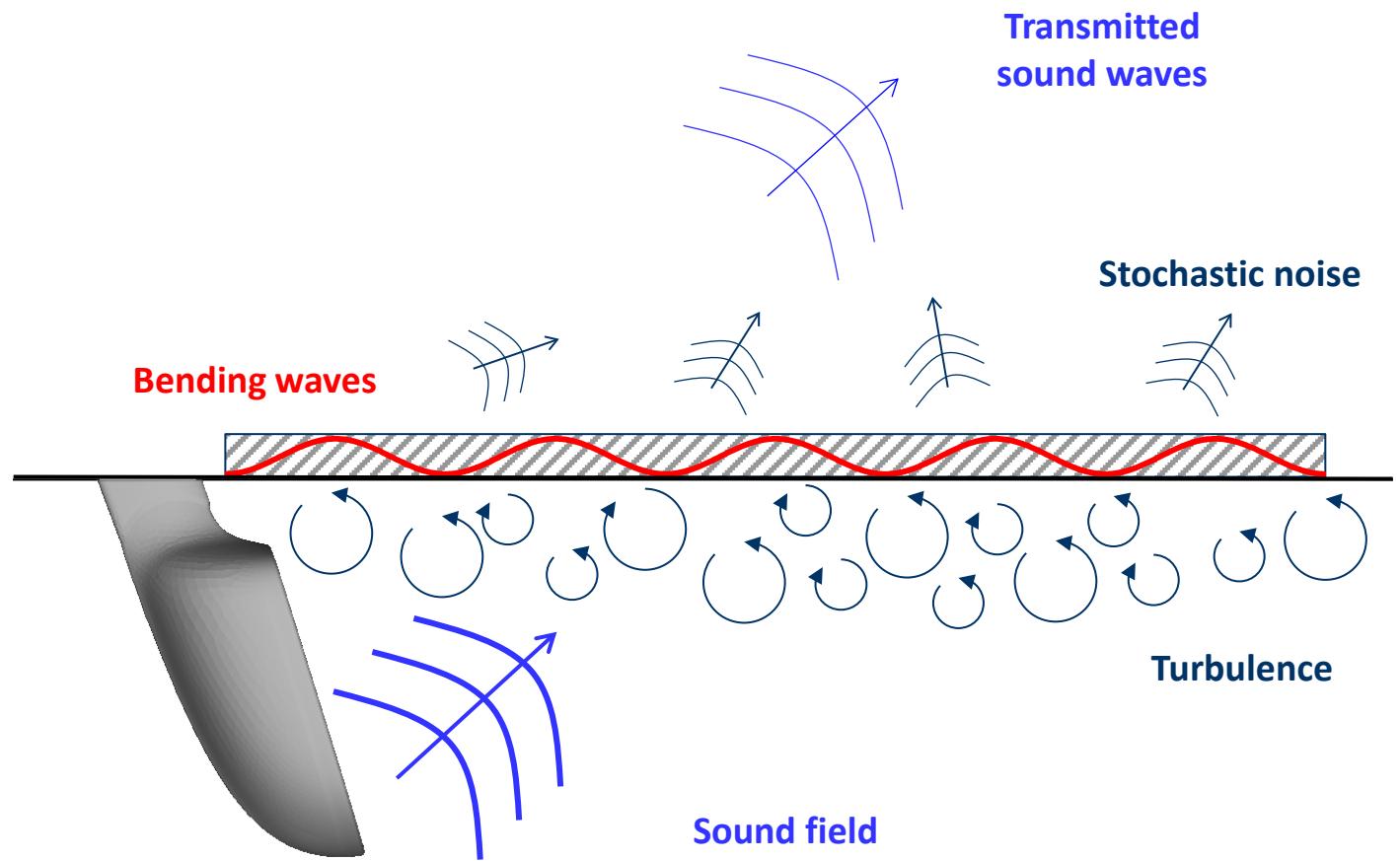
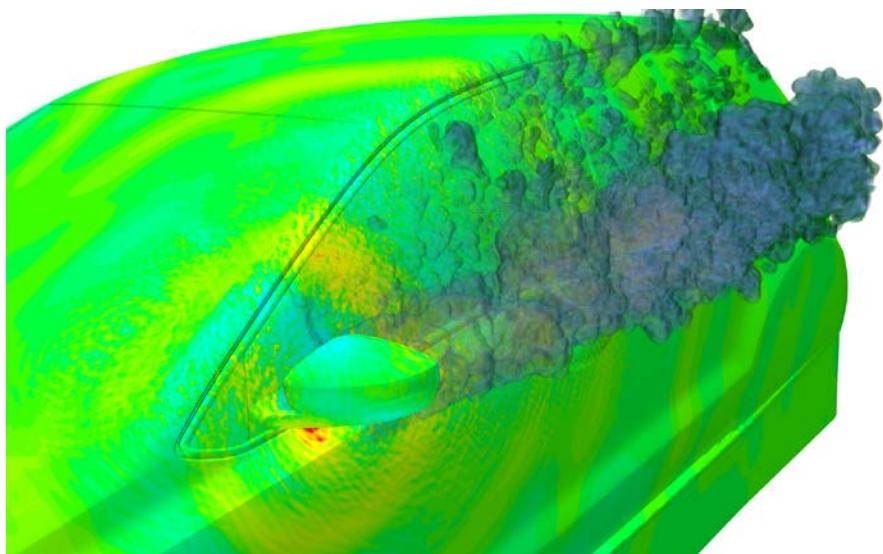
$$\vec{R}(\vec{u}, q) = \begin{cases} R^q = \nabla \cdot \vec{u} = 0 \\ \vec{R}^u = -\frac{\partial \vec{u}}{\partial t} - (\vec{v} \cdot \nabla) \vec{u} - \nabla \vec{u} \cdot \vec{v} - \nabla \cdot \boldsymbol{\tau}^a + \nabla q = \vec{0} \end{cases}$$

## Sensitivity Derivatives:

$$\frac{\delta L}{\delta b} = \int_S -\nu \frac{\partial \vec{u}}{\partial n} \cdot \frac{\partial \vec{v}}{\partial n} + [...] dS$$

# PHYSICAL MECHANISMS OF SOUND PROPAGATION TO THE INTERIOR

1. Noise creation
2. Sound propagation to window
3. Structural vibration
4. Noise radiation in the interior



## PROCESS FOR INTERIOR NOISE PREDICTION

### 1. Noise creation (time resolved CFD)

- ▶ OpenFOAM 2.1.1
- ▶ IDDES Spalart Allmaras
- ▶ SnappyHexMesh for meshing

$$R^p = \nabla \cdot \vec{v} = 0$$

$$\vec{R}^v = \frac{\partial \vec{v}}{\partial t} + (\vec{v} \cdot \nabla) \vec{v} - \nabla \cdot \tau + \nabla p = \vec{0}$$

Hydrodynamic pressure  
 $p$  on the mirror

### 2. Noise radiation (Kirchhoff-Integral)

- ▶ OpenFOAM 2.1.1
- ▶ Fully parallelised

$$p' = \int_{\text{mirror}} \left[ \frac{1}{r^3} (\vec{r} \cdot \hat{n}) p + \frac{1}{cr^2} (\vec{r} \cdot \hat{n}) \frac{\partial p}{\partial t} \right] dS$$

### 3. Structure vibration (Bending waves)

- ▶ OpenFOAM 1.6-ext
- ▶ Using finiteArea solver

$$R^w = \frac{\partial^2 w}{\partial t^2} + \eta_1 \frac{D}{m'} \nabla^4 w + \eta_2 \frac{\partial w}{\partial t} + \eta_3 \sqrt{\frac{D}{m'} \frac{\partial}{\partial t}} \nabla^2 w - \frac{p'}{m'} = 0$$

### 4. Sound propagation (Wave equation)

- ▶ OpenFOAM 1.6-ext
- ▶ Castellated mesh in vehicle interior

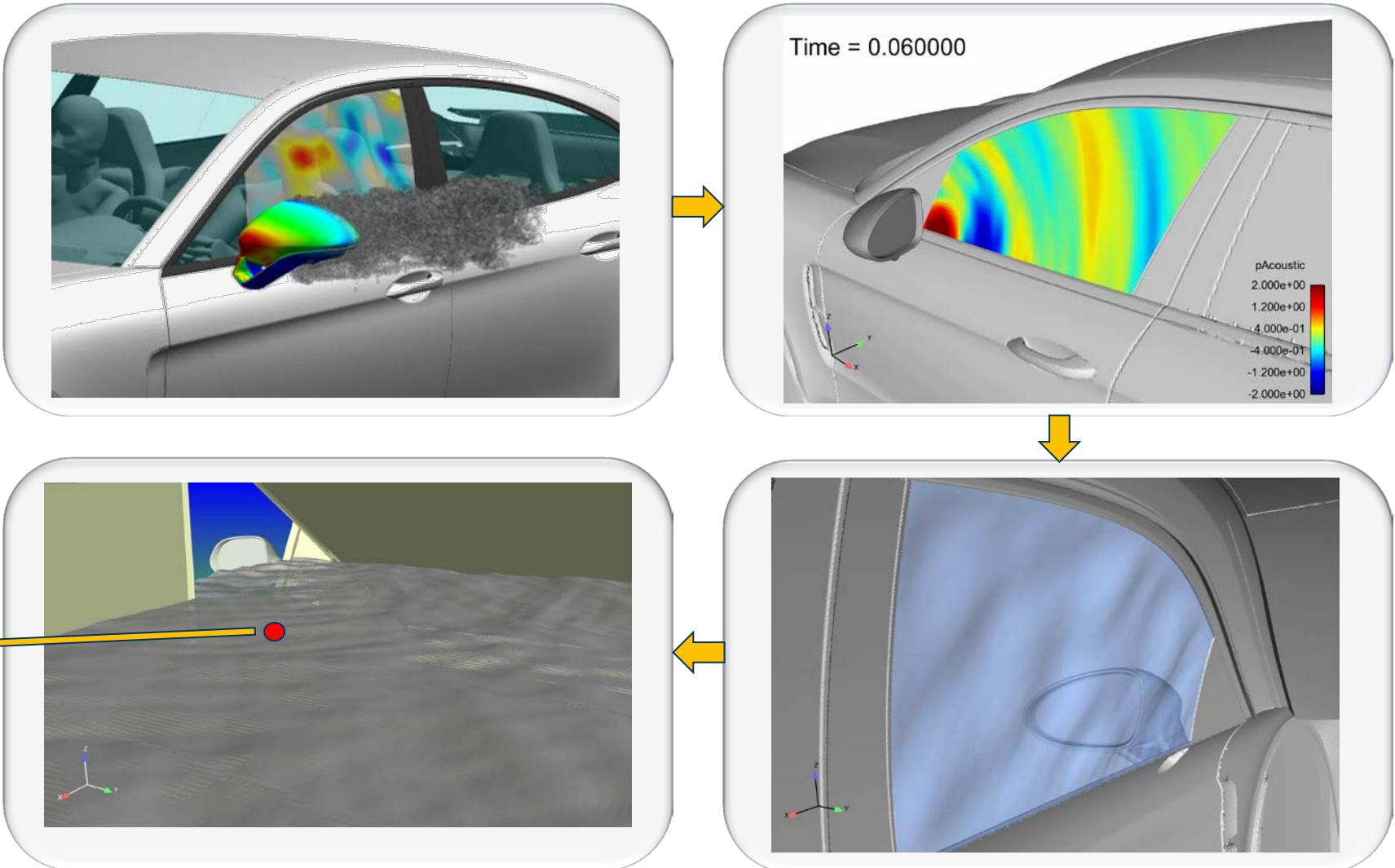
$$R^{p^a} = \frac{\partial^2 p^a}{\partial t^2} + \nabla^2 p^a = 0$$

BC on the window:

$$\frac{\partial p^a}{\partial n} - \rho \frac{\partial^2 w}{\partial t^2} = 0$$

## WIND NOISE PREDICTION

1. Turbulent Flow
2. Sound Radiation
3. Structural Vibration
4. Cabin Noise



# ADJOINT PROCESS FOR SENSITIVITIES COMPUTATION

1. Noise creation (time resolved CFD)



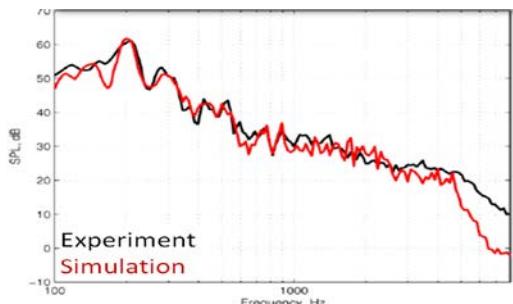
2. Noise radiation (Kirchhoff-Integral)



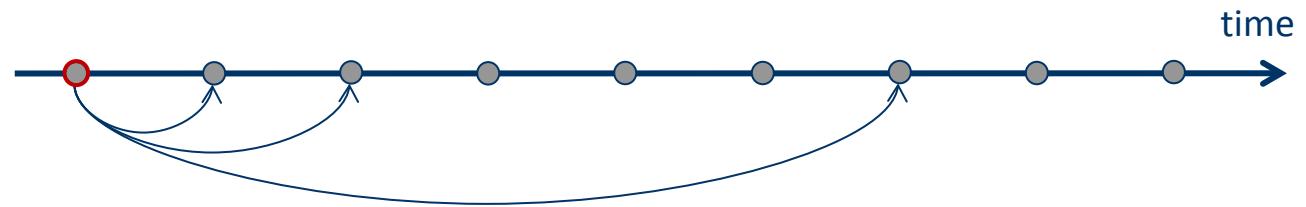
3. Structure vibration (Bending waves)



4. Sound propagation (Wave equation)



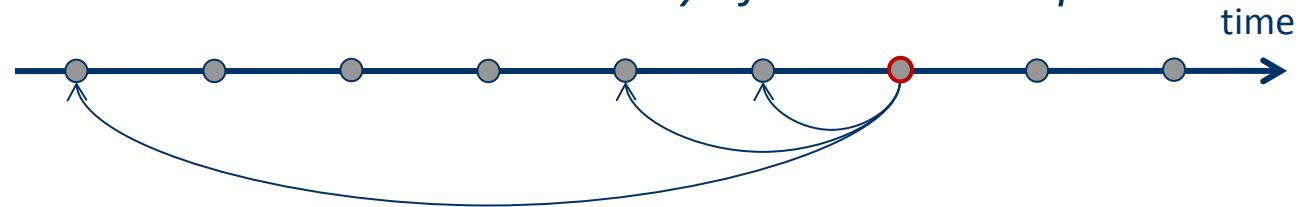
## Primal simulation



*Each time step has an effect on the following time steps*

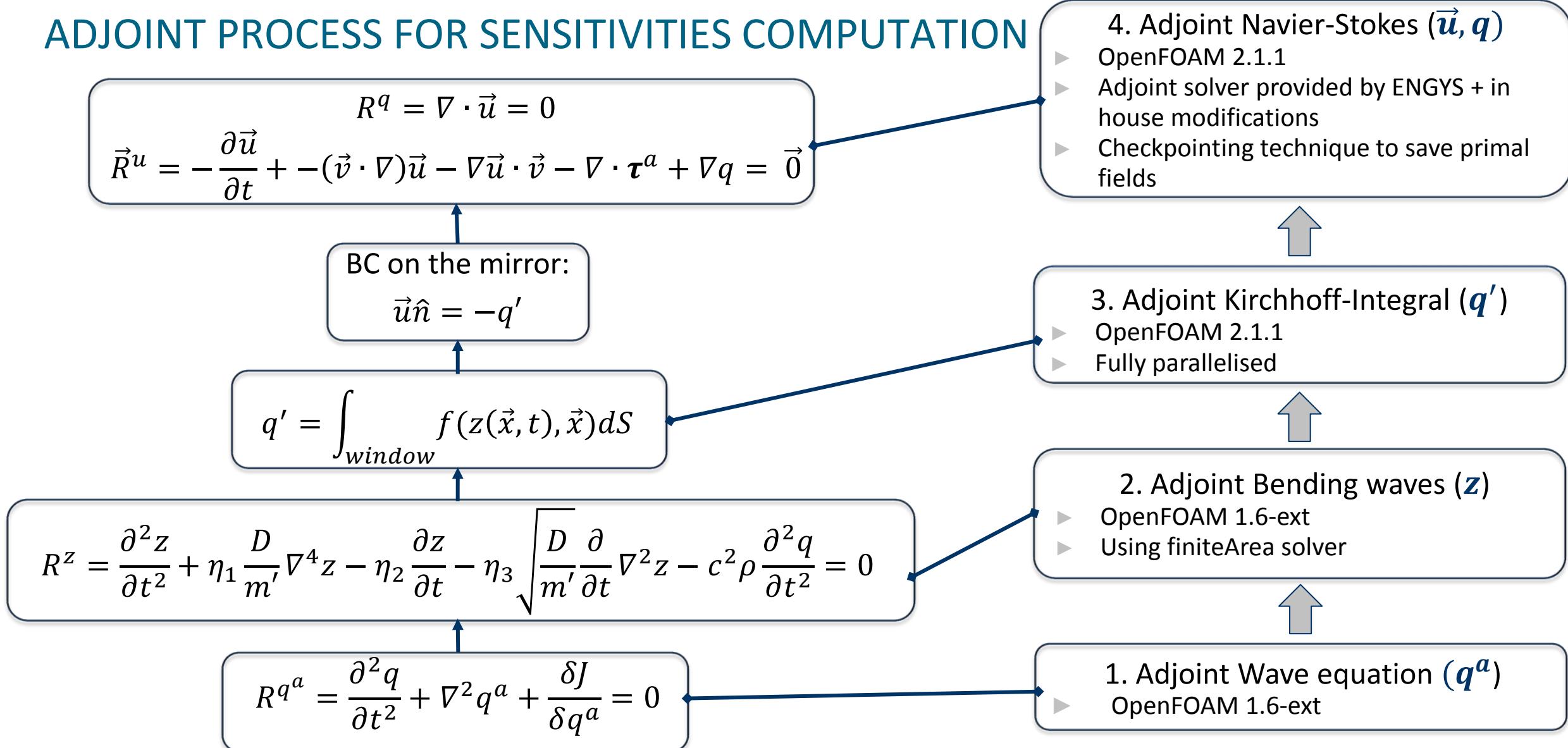
## Adjoint simulation

*What is the sensitivity of each time step?*

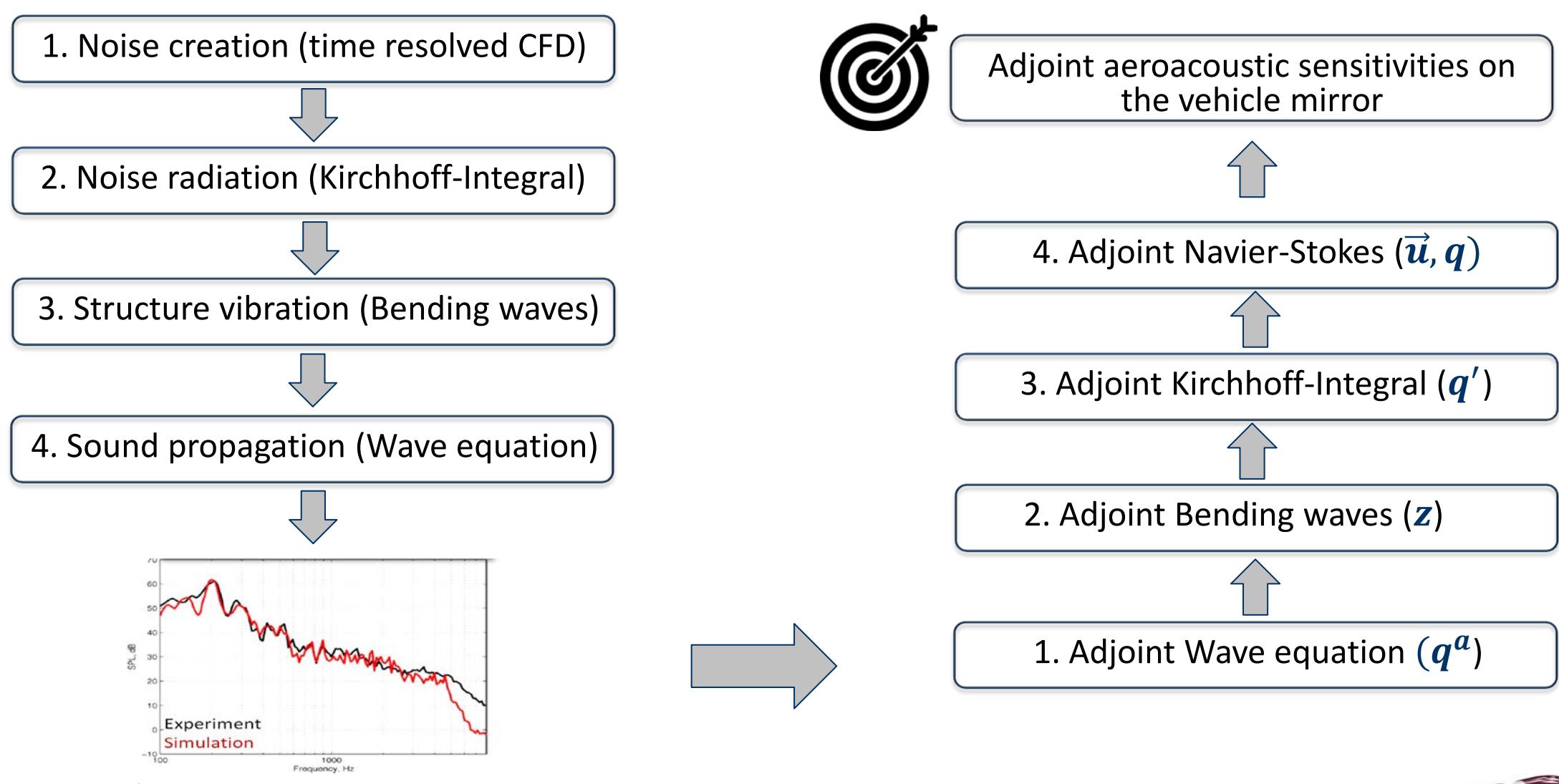


*The information of this sensitivity has to travel backwards in time*

# ADJOINT PROCESS FOR SENSITIVITIES COMPUTATION

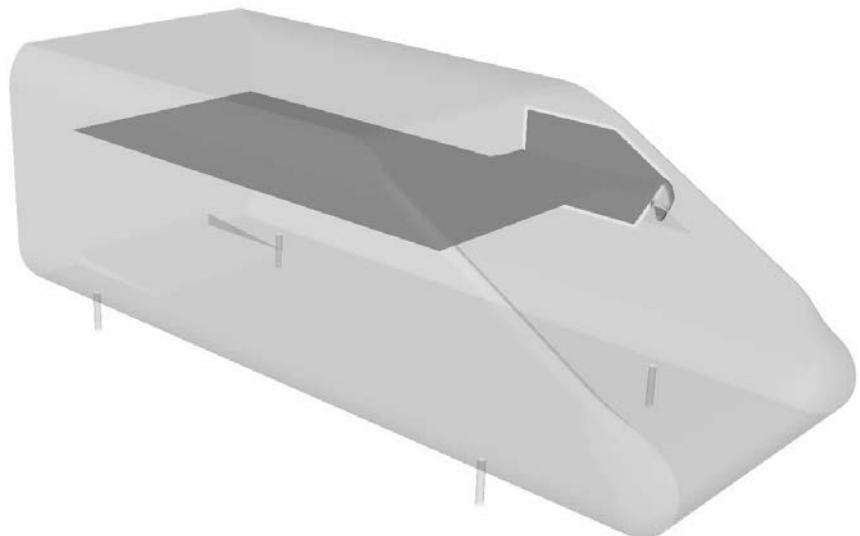


# ADJOINT PROCESS FOR SENSITIVITIES COMPUTATION

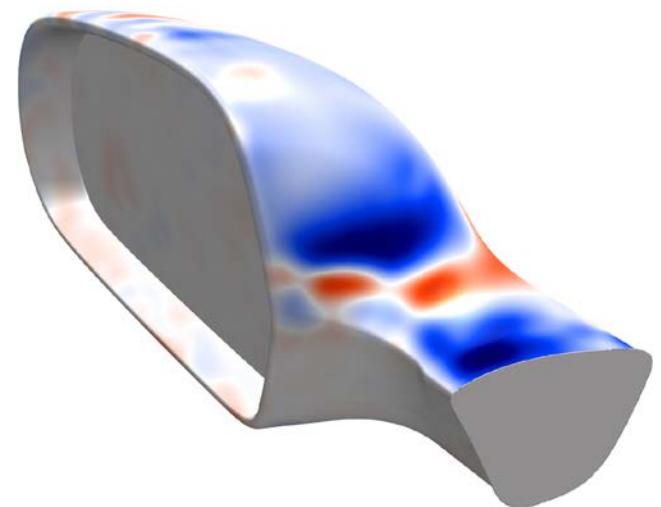


## ADJOINT AEROACOUSTIC SENSITIVITIES

Simulation of the adjoint  
aeroacoustic chain



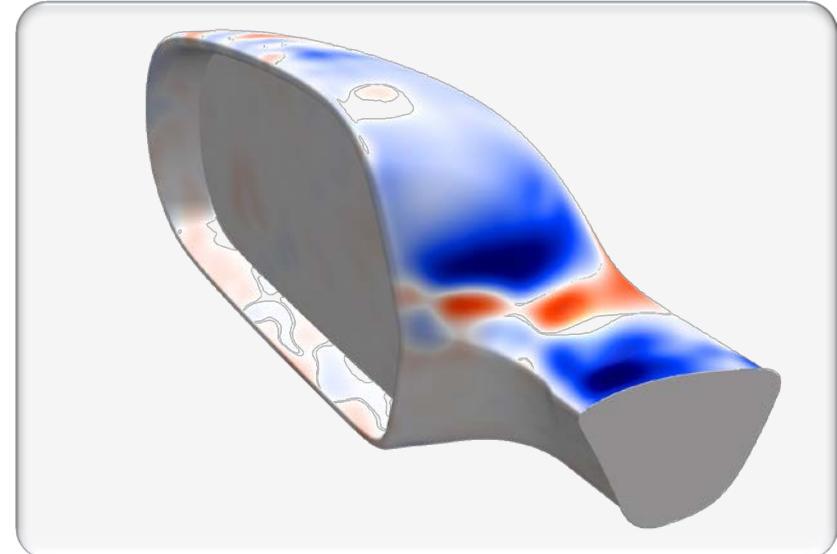
**Red:** push in to reduce interior noise  
**Blue:** pull out to reduce interior noise



## ADJOINT AEROACOUSTIC SENSITIVITIES

Local Morphing  
→  $\approx 8\%$  reduction

Red: push in to reduce interior noise  
Blue: pull out to reduce interior noise



## CONCLUSIONS & FUTURE STEPS

- Continuous adjoint method for interior noise prediction developed and implemented in OpenFOAM
  - including all steps of noise propagation from the flow to the interior
- One-step optimisation performed as proof of concept
- Solutions for the bottlenecks of data handling and computational cost are under consideration
- A complete optimization process will be performed in the future

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<http://ioda.sems.qmul.ac.uk>

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